

## ChE 374–Lecture 25–Integral Momentum Balance

- Integral Momentum Balance
  - Find Forces or Accelerations
- $\sum \vec{F} = \frac{\partial}{\partial t} \int_{CV} \rho \vec{v} dV + \int_{CS} \rho \vec{v} \vec{v} \cdot \vec{n} dA$ .
  - For uniform properties:
$$\sum \vec{F} = \frac{d}{dt} (m\vec{v}) + (\sum \dot{m}\vec{v})_{out} - (\sum \dot{m}\vec{v})_{in}$$
  - For uniform properties and Steady State:
$$\sum \vec{F} = (\sum \dot{m}\vec{v})_{out} - (\sum \dot{m}\vec{v})_{in}$$
    - \* TAKE THIS EQUATION LITERALLY.
      - Vector components are positive in the positive coordinate direction.
      - $\dot{m} = \rho A |v|$  is a scalar quantity, always positive.
  - Note:  $\dot{m}\vec{v}$  is momentum flow rate.
  - Note:  $\rho \vec{v} \vec{v}$  is momentum flux.
  - Note:  $\rho v_x \vec{v}$  is x-momentum flux.
- Choose a control volume
  - Not limited to the fluid
  - Choose it perpendicular to inlets and outlets
  - The momentum balance is WITH RESPECT TO THIS CV.
    - \* So velocities are RELATIVE to the control volume.
    - \* Also,  $\dot{m}$  is also relative to the control volume.
- Forces are (1) BODY; (2) SURFACE; (3) OTHER
  - Body forces are gravity ( $mg$ ).
  - Surface forces are Pressure (always normal to CV, and towards the CV); and viscous forces (usually neglect at inlets and outlets where the flows often cross the boundary).
  - Other forces are external, like bolts, the ground, and other anchoring forces, etc.
- Momentum flux correction factor corrects for nonuniform flow at inlets and outlets. Similar to kinetic energy correction factor.
$$\beta \bar{v} = \frac{1}{A} \int v^2 dA. \quad \beta = 4/3 \text{ for laminar}$$
$$\alpha \bar{v} = \frac{1}{A} \int v^3 dA. \quad \alpha = 2 \text{ for laminar}$$
- Example: Flow through a nozzle.
  - Restraining force is  $F = A_1(P_1 + \rho v_1^2(1 - A_1/A_2))$ , where (1) is inlet, (2) is outlet, and  $F$  is in the direction opposite the flow.
- Example: Flow deflected 90 degrees:
  - Restraining force is  $F = \dot{m}v$ , with force directed opposite the flow.
- Example: Flow deflected 180 degrees:
  - Restraining force is  $F = 2\dot{m}v$ , with force directed opposite the flow.
- Example: Flow deflected  $\theta < 90$  degrees:
$$F_x = \dot{m}v(1 - \cos \theta), \text{ x is flow inlet direction, and } F_x \text{ is opposite the flow direction.}$$
$$F_y = \dot{m}v(\sin \theta), \text{ y is perpendicular to inlet direction, and } F_y \text{ is with the direction of deflection.}$$

# Class 25 - Momentum Balance.

Where are we?

- Fluid Statics
- Mass Balance
- Energy Balance
- \* Momentum Balance.  
Chp 6.1-6.4 (Linear Momentum)

• Velocity Profiles and Forces  $\rightarrow$  Mom. Bal.

$$\underline{F = ma = m \frac{dv}{dt} = \frac{d(mv)}{dt} = \text{Rate of Momentum.}}$$

• Derived Laminar pipe velocity using a Force (Momentum) Balance.

- Used a Differential Approach.

\* - Here, Do Integral Approach, Then Discuss (Later Class) Differential Balances formally.

Goal: Find  $\vec{F}$ ,  $\vec{a}$ ,  $\vec{v}$ , given forces, or  $\vec{v}$ , etc.

## Reynolds Transport Theorem.

$$\frac{dB_{\text{sys}}}{dt} = \frac{\partial}{\partial t} \int_{\text{CV}} \rho b dV + \int_{\text{CS}} \rho b \vec{v} \cdot \vec{n} dA \quad ; \quad B = m\vec{v}, \quad b = \vec{v}$$

$$* \quad \sum \vec{F} = \frac{\partial}{\partial t} \int_{\text{CV}} \rho \vec{v} dV + \int_{\text{CS}} \rho \vec{v} \vec{v} \cdot \vec{n} dA$$

Uniform Properties:

$$* \quad \sum \vec{F} = \frac{d}{dt} (m\vec{v}) + (\sum \dot{m} \vec{v})_{\text{out}} - (\sum \dot{m} \vec{v})_{\text{in}}$$

0 at S.S. ( $\vec{v}$  here is  $\vec{v}$  of C.V.  $\neq \vec{v}_{\text{out}}, \vec{v}_{\text{in}}$ )

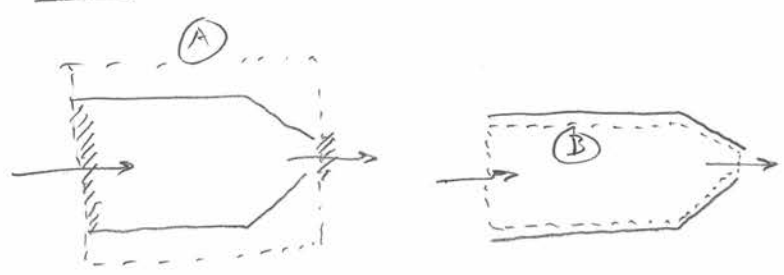
- Vector Eq.  $\rightarrow$  Split into x, y components.

Note:  $\dot{m} \vec{v}$  is momentum rate. :  $\text{kg} \cdot \text{m/s}^2$

$\rho \vec{v} \vec{v}$  is momentum flux :  $\text{kg/m} \cdot \text{s}^2$

# Control Volume

- we've implied the use of a C.V.
- velocities are relative to the C.V. ;  $v = V - v_{cv}$
- Be Smart:



- (A) → ignore the details inside → easier
- (B) → must account for cell forces of the nozzle, fluid, → harder

- Choose C.V.  $\perp$  to inlet/outlet.
- C.V. is not limited to fluid
  - in (A), only have nonzero x-forces in the shaded region.

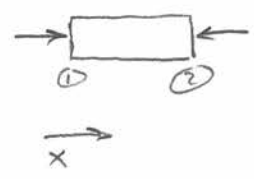
## $\Sigma \vec{F}$ Terms

- These are external forces
  - ① Body Forces
  - ② Surface Forces (fluid)
  - ③ Other

① Body Forces = Gravity =  $m\vec{g} = -\rho V \vec{g} = F_y$

② Surface Forces = Pressure  
Viscous (Ignore, usually)

•  $\vec{F}_p = -P\vec{n}$



$F_{p,x} = P_1 A - P_2 A$   
+ in + x Dir

### ③ Other

- Bolts, Ground, Anchoring forces, etc.
- $\vec{F}_o$  is the force of these on the C.V.
- Careful of the sign
- Ground exerts upward force;



Bolts pull ←

### Example 1

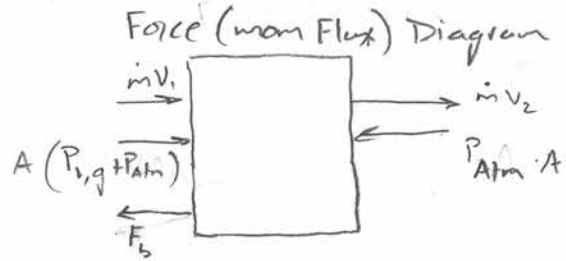
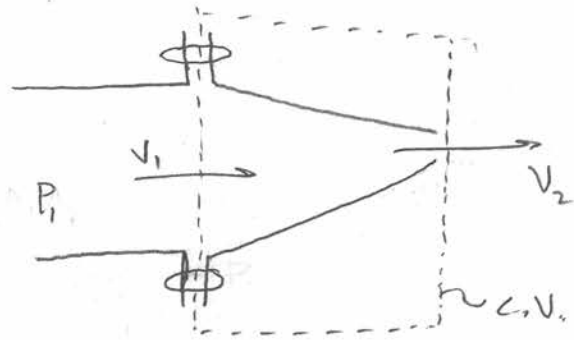
- Flow through Nozzle
- S.S.

\* Find Force on Bolts,  
Given  $P_1, V_1, A_1, A_2$

Governing Eq.

$$\left( \sum F \right) = \left( \frac{dm\vec{V}}{dt} \right) + (\dot{m}\vec{V})_{out} - (\dot{m}\vec{V})_{in}$$

- $\dot{m} = \rho A |V|$
- Forces from  $P$ , Bolts



$$\left( -F_b + A(P_{1,g} + P_{Atm} - P_{Atm}) \right) = (0) + \rho A_2 V_2^2 - \rho A_1 V_1^2$$

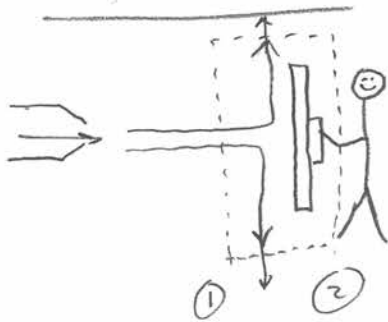
$$\bullet V_1 A_1 = V_2 A_2$$

$$F_b = (P_{1,g} \cdot A_1 + \rho A_1 V_1^2 - \rho A_2 \left( \frac{V_1 A_1}{A_2} \right)^2)$$

$$* F_b = A_1 \left[ P_1 + \rho V_1^2 \left( 1 - \frac{A_1}{A_2} \right) \right]$$

Note:  $P_{atm}$  acts on all surfaces  
 → Cancels  
 → Work in Gage Pressure

### Example 2



- Flow turns, Find holding force
- S.S.

Y-Direction → Cancels

X-Direction

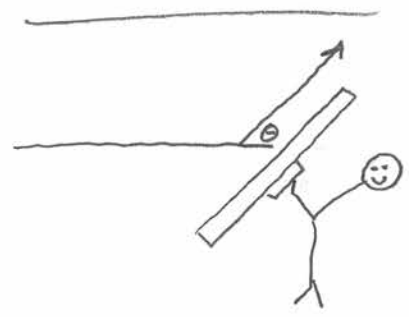
- Atm Cancels

$$\sum \vec{F} = \frac{d\vec{mV}}{dt} + \dot{m}\vec{V}_{out} - \dot{m}\vec{V}_{in}$$

$$-F_{arm} = -\dot{m}V_1$$

$$F_{arm} = \dot{m}V_1$$

Example 3



- Incline
- SS
- Find Force

$$\Sigma \vec{F} = \frac{d\vec{p}}{dt} + (m\vec{v})_0 - (m\vec{v})_i$$

X-momentum:  $-F_x = mV \cos \theta - mV$

Y-momentum:  $F_y = mV \sin \theta$

$$\begin{cases} F_x = mV(1 - \cos \theta) \\ F_y = mV \sin \theta \\ \vec{F} = -F_x \vec{i} + F_y \vec{j} \end{cases}$$

Note: Could have written x- as:

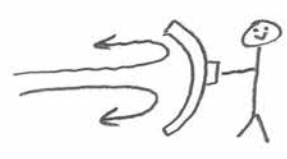
$$\begin{aligned} \rightarrow F_x &= mV \cos \theta - mV \\ &= mV(\cos \theta - 1) \text{ is negative so force acts } \leftarrow \end{aligned}$$

If Draw  $F_x \leftarrow$ , and sign implies  $\leftarrow$ , then a Pos  $F_x$  is  $\leftarrow$

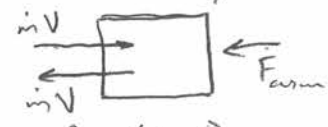
If Draw  $F_x \rightarrow$ , and sign implies  $\rightarrow$ , then a Pos  $F_x$  is  $\rightarrow$

If Confused, Always Draw  $F_x$  in + X Dir. Then  $+F_x \rightarrow$   
 $-F_x \leftarrow$

Example 4



S.S. X-only  $\Sigma \vec{F} = \frac{d\vec{p}}{dt} + (m\vec{v})_0 - (m\vec{v})_i$



$$-F_{arm} = (-mV) - (mV)$$

$$F_{arm} = 2mV = 2x \Rightarrow !$$

- See All Book Examples
- What if unsteady?
- Be careful about interpreting Thrust in Book:

$$\sum \vec{F}_{ext} = \frac{dm\vec{V}}{dt} + \dot{m}\vec{V}_{out} - (\dot{m}\vec{V}_{in})$$

$$0 \Rightarrow \frac{dm\vec{V}}{dt} = (\dot{m}\vec{V})_{in} - (\dot{m}\vec{V})_{out}$$

Book calls this  $\vec{F}_{thrust}$ , but it is just the unsteady acceleration term, driven by the rate of momentum transfer.

- Know what to Do if not uniform

→ Backup and integrate:

$$S.S. : \sum \vec{F} = \int_{c.s.} \rho \vec{V} V_{out} dA - \int_{c.s.} \rho \vec{V} V_{in} dA$$