

## ChE 374–Lecture 13–Mechanical Energy Analysis of Steady Flow

- Kinetic Energy Correction Factor
  - $\frac{v^2}{2}$  term appears in energy equations
  - Usually assume uniform  $v$ .
  - In general  $v$  is nonuniform (e.g., pipes have  $v=0$  at the walls).
    - \* For mass flow, just use average velocity, then can treat as uniform.
    - \* For K.E. this does not work since  $\bar{v}^2 \neq \overline{v^2}$
  - Correct with a fudge factor,  $\alpha$ :  $K.E. = \frac{1}{2}\alpha\bar{v}^2$ .
  - $\alpha = 2$  for laminar flow and  $\alpha = 1.04 - 1.11$  for turbulent flow.
  - Often ignored:
    - \* Most flows are turbulent
    - \* K.E. is small compared to  $\Delta P$ , or  $\Delta z$ .
    - \* Small error compared to other assumptions or unknowns.
- Friction and Mechanical Energy Losses

- SS Energy Equation:

$$\dot{Q} + \dot{W}_s = \dot{m}\Delta u + \dot{m} \underbrace{\left( \frac{\Delta P}{\rho} + \frac{\Delta v^2}{2} + g\Delta z \right)}_{\Delta e_{mech}}.$$

- We've been ignoring  $\dot{Q}$ ,  $\dot{W}_s$ ,  $\Delta u$ .
- Now, account for  $\dot{W}_s$  and “Losses”.
- Observations:
  - 1 Take  $\dot{Q} = \dot{W}_s = \Delta u = 0 \rightarrow \Delta e_{mech} = 0 \rightarrow e_{mech}$  is conserved.
  - 2 Consider a heated pipe, no friction, then  $\dot{m}\Delta u = \dot{Q}$ .
    - So that heat  $\rightarrow \Delta u$
  - 3 Friction converts mechanical energy to internal energy.
    - $\Delta u$  not from heat transfer is from friction losses. Rearrange Energy Eqn:

$$\dot{m} \left( \frac{\Delta P}{\rho} + \frac{\Delta v^2}{2} + g\Delta z \right) = \dot{W}_s + \underbrace{(\dot{Q} - \dot{m}\Delta u)}_{-F}.$$

- $F$  is +.
- $F$  decreases  $e_{mech}$ .
- Book calls this  $\dot{E}_{mech, loss}$ .
- 4 Head Form:
 
$$\frac{\Delta P}{\rho g} + \frac{\Delta v^2}{2g} + \Delta z = h_w - h_L$$
  - with  $h_L = F/\dot{m}g$ ,  $h_w = \dot{W}_s/\dot{m}g$ .
  - Normally  $F$  is friction in pipes, with losses in pumps accounted with efficiency.

- Examples:
  - Example 1: Raise a liquid:  $h_w = \Delta z + h_L$ .
  - Example 2: Pump a liquid:  $h_w = \frac{\Delta P}{\rho g} + h_L$ .
  - Example 3: Nozzle:  $-\frac{\Delta P}{\rho g} = \frac{\Delta v^2}{2g} + h_L$ .
  - In each case,  $h_L$  is wasted:  $\Delta z$ ,  $\Delta P$ , and  $\Delta v$  are less than without loss.
  - Text 5-94.
- Note: one equation, one unknown:  $(\dot{m}, \Delta P, \Delta v^2, \Delta z, \dot{W}_s, \eta, F)$ .
  - Set all but one and solve for that one. CONSIDER THIS IN BOOK EXAMPLES, HW.

# Lecture 13 - Mechanical Energy Analysis of Steady Flow

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Items: Class Project

K.E. correction factor  $\alpha$

Friction / Loss

Examples.

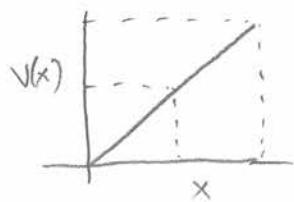
K.E. correction factor.

- $\frac{V^2}{2}$  term appears
- Been assuming uniform  $V$
- In general  $V \neq$  uniform:

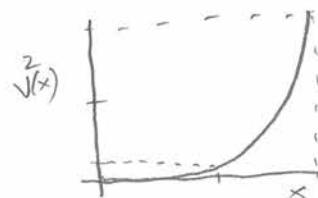
Pipes



- For mass flow, just use  $\bar{V}$
- But for K.E. This Does not work:  $\bar{V}^2 \neq \overline{V^2}$



$$\text{linear} \rightarrow \bar{V} = V(\bar{x})$$



$$\text{nonlinear} \rightarrow \bar{V}^2 \neq (V(\bar{x}))^2$$

- Correct w/ a fudge factor:  $KE = \frac{1}{2} \alpha \bar{V}^2$

\*  $\alpha = 2$  for laminar pipe

$$\alpha = 1.04 - 1.11 \text{ for Turb. flow.}$$

- often ignored.

- most flows turb

- KE small vs P or h

- Small relative to other assumptions.

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## Friction / Losses

SS Energy Equation

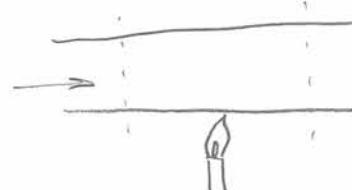
$$\dot{Q} + \dot{W}_s = \dot{m} \Delta u + \dot{m} \underbrace{\left( \frac{\Delta P}{\rho} + \frac{\Delta V^2}{2} + g \Delta z \right)}_{\Delta e_m}$$

- Been ignoring  $\dot{Q}$ ,  $\dot{W}_s$ ,  $\Delta u$
- Now, add in  $\dot{W}_s$ , "Losses"

### Observations

1.  $\dot{Q} = \dot{W}_s = \Delta u = 0 \rightarrow \Delta e_m = 0 \rightarrow e_m \text{ conserved}$   
- B.E.

2. Heater, no friction,  
 $\dot{m} \Delta u = \dot{Q}$



Heat  $\rightarrow \Delta u$

3. Friction converts mechanical energy to internal energy  
 $\Delta u$  not from heat transfer is friction loss ↑

$$\dot{m} \left( \frac{\Delta P}{\rho} + \frac{\Delta V^2}{2} + g \Delta z \right) = \dot{W}_s + \underbrace{(\dot{Q} - \dot{m} \Delta u)}_{-\dot{H}}$$

•  $\dot{H}$  is +

• Decreases  $e_m$

• Book calls this  $\dot{E}_{mech, loss}$ .

4. Head Form:

$$\frac{\Delta P}{\rho g} + \frac{\Delta V^2}{2g} + \Delta z = h_a - h_L \quad ; \quad h_L = \frac{\dot{H}}{\dot{m} \cdot g}$$

- normally this is Friction in pipes, with losses in pumps accounted with efficiency

### Example - Raise a Liquid

$\Delta z:$



$$\cancel{\frac{\Delta P}{\rho g}} + \cancel{\frac{\Delta V^2}{2g}} + \Delta z = h_w - h_L$$

$$* h_w = \Delta z + h_L$$

### Example - Pump a Liquid.

$\Delta P:$



$$\frac{\Delta P}{\rho g} + \cancel{\frac{\Delta V^2}{2g}} + \cancel{gz} = h_w - h_L$$

$$* h_w = \frac{\Delta P}{\rho g} + h_L$$

### Example - Nozzle

$\Delta V$



$$\frac{\Delta P}{\rho g} + \cancel{\frac{\Delta V^2}{2g}} + \cancel{gz} = h_w - h_L$$

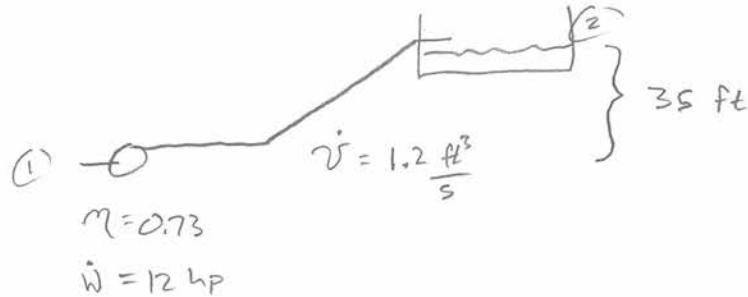
- In each case  $h_L$  is wasted,  
 $\Delta z$ ,  $\Delta P$ ,  $\Delta V$  is less!

$$-\frac{\Delta P}{\rho g} = \frac{\Delta V^2}{2g} + h_L$$

- $(W_s - F) = W_u = \text{inden}$ ;  $W_u$  is the useful work!

Example

Text S-94



- Find  $h_L$ ,
- Find Power to overcome it.

**T.P.S.**

→ Q: where to start? → E.E. → write it.

$$w_s - F = m \left( \frac{P_2}{\rho} + \frac{V_2^2}{2} + g z_2 \right) - m \left( \frac{P_1}{\rho} + \frac{V_1^2}{2} + g z_1 \right)$$

- Pick 2 points: - with  $w_s$ ,  $F$  the locations matter
  - go w/ the flow
  - $w_s$  raises  $z_1$  → 2-1 is top-bot
  - else get wrong signs.

• Simplify:  $V_1 = V_2 = 0$        $P_1 = P_2 = P_{atm}$   
 $z_2 = h$        $m_i = m_o = C$

→  $w_s - F = mgh$

• Head Form  $\div mgh$

$$\rightarrow \frac{w_s}{mgh} - h_L = h$$

Terms:  $w_s = ? = \eta W = 8.76 \text{ hp}$

$$m = \rho V = (62.3 \text{ lbm}/\text{ft}^3)(1.2 \text{ ft}^3/\text{s}) = 74.76 \text{ lbm/s}$$

$$g = 32.2 \text{ ft/s}^2$$

$$h = 35 \text{ ft}$$

$$\rightarrow h_L = 29.4 \text{ ft}$$

$$\frac{w_s}{mgh} = \frac{h_L}{h+h_L}$$

$$\text{Power}_L = mgh_L = [4.0 \text{ hp}]$$

$$\rightarrow \underline{w_s = 4.0 \text{ hp}}$$

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- Note: we could reverse all this too  $\rightarrow$  Dam Power.
- Note 1 Eqn, 1 Unknown.

$$\begin{pmatrix} m \\ \Delta P \\ \Delta V^2 \\ \Delta z \\ W_s \\ \eta \\ F \end{pmatrix}$$

- Set or assume all but one solve.
- All prob are like this.
- Use when reading book examples.
- Note how to convert between head forms.